

# BOUNDARY ELEMENTS IN POTENTIAL AND ELASTICITY THEORY

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**Abstract**—A general theory that describes the B.I.E. linear approximation in potential and elasticity problems, is developed. A method to treat the Dirichlet condition in sharp vertex is presented. Though the study is developed for linear elements, its extension to higher order interpolation is straightforward.

A new direct assembling procedure of the global of equations to be solved, is finally showed.

## 1. INTRODUCTION

Since the publication of Galileo's "Dialogue of two new sciences", which established the basis for the scientific approach to building problems there has been no landmark as the creation of electronic digital computer. Effectively, the creation of the "Ecole des Ponts et Chaussées" and consequently the appearance of was no more than the development of the first "engineers" positive ideas contained in Galileo's book. The erection of the larger, better and cheaper works demanded by the nineteenth century industrial revolution, showed clearly the advantages of a scientific viewpoint over traditional empirical methods.

On its part, the computer has removed the traditional human limitations, and its impact has produced a stimulating reexamination of established knowledge in every field of science.

In structural analysis the well known 1956 paper of Turner *et al.* [1] opened one as striking way, that several other interesting numerical methods nearly fell forgotten under the splendid F.E.M. boom.

The simplicity of the method, perfectly suited to computer, attracted many researchers whose work has converted F.E.M. in an everyday tool of engineering profession.

Other methods were circumvented and only the unsurmountable difficulties for handling some problems with F.E.M. have produced the recent interest in several alternatives to it.

One of these possibilities is the Boundary Integral Equation Method (B.I.E.M.). The first idea was presented as early as 1963 by Jaswon [2] in a paper over potential problems; Rizzo [3] successfully extended the method to more complicated problems and Cruse *et al.* have shown in several papers [4-9] how the new method can cover the "misprints" of F.E.M. Several other authors have computer refinements [10], coupling B.I.E.M. with F.E.M. [11], etc. Also the first attempt to produce a book on the topic has occurred [12] and two International Conferences specifically dedicated to B.I.E.M. [13, 14] have been held.

Nevertheless the mastering of the method is far from being reached and further knowledge of computer and numerical techniques are still needed to make B.I.E.M. as powerful a tool as F.E.M. is. This paper tries to be a little contribution in that way.

The implementation of B.I.E.M. offers the same

difficulty when the isoparametric element concept is applied to boundary domains with sharp corners.

Standard procedures [10, 15] produce important errors if the size of the elements near the vertex is not small enough. Some authors [15] have proposed an alternative procedure in which they define in the corner as many nodes as elements are interconnected.

The drawback of both lines of action is clear; the first one multiplies the number of elements, losing one of the more outstanding features of the isoparametric representation. The latter increases the number of equations to be solved, which by the special character of B.I.E.M. is highly undesirable. Then this is the problem area we shall address.

## 2. B.I.E.M. TECHNIQUE IN POTENTIAL THEORY

As it is well known, the B.I.E.M. is based upon the existence of two items: a fundamental solution of the field equation and a reciprocity theorem which, in potential theory, is the Green formula [16].

Given two functions  $\psi$  and  $\phi$  with  $C^2$ -continuity in a bounded region  $D$  and  $C^1$  in  $D + \partial D$ , where  $\partial D$  is a piecewise smooth boundary, Green formula is

$$\int_D (\psi \nabla^2 \phi - \phi \nabla^2 \psi) dv = \int_{\partial D} \left( \psi \frac{\partial \phi}{\partial \nu} - \phi \frac{\partial \psi}{\partial \nu} \right) ds. \quad (1)$$

If

$$\begin{aligned} -\nabla^2 \phi &= 4\pi \delta(p, q) \\ \nabla^2 \psi &= 0 \end{aligned} \quad (2)$$

being  $p, q$  two points in  $D$ , and  $\delta(p, q)$  the Dirac function, one obtains the representation

$$\psi(P) = \frac{1}{4\pi} \int_{\partial D} \frac{1}{r} \frac{\partial \psi}{\partial \nu} ds - \frac{1}{4\pi} \int_{\partial D} \psi \frac{\partial}{\partial \nu} \left( \frac{1}{r} \right) ds. \quad (3)$$

For two dimensions the fundamental solution to be used is  $1/2\pi \ln(1/r)$ .

The particularisation of eqn (3) for points in the boundary leads to (see Ref. [16], vol. II, pp. 256-257)

$$257) c\psi(P) + \int_{\partial D} \psi \frac{d\Delta}{d\nu} ds = \int_{\partial D} \Delta \frac{d\psi}{d\nu} ds \quad (4)$$

where in 3D

$$\Delta = \frac{1}{r} \quad c = \begin{cases} 4\pi & \text{for } P \text{ in } D \\ 2\pi & \text{for } P \text{ on } D \\ 0 & \text{for } P \text{ outside } D. \end{cases}$$

or in 2D

$$\Delta = \ln \frac{1}{r} \quad c = \begin{cases} 2\pi & \text{for } P \text{ in } D \\ \pi & \text{for } P \text{ on } D \\ 0 & \text{for } P \text{ outside } D. \end{cases}$$

if there is a continuous tangent at  $P$ . If there is a conical vertex,  $C$  is the corresponding angle.

Equation (23) is now discretized following the isoparametric F.E. idea of Irons.

Both variables ( $\psi, \partial\psi/\partial\nu$ ) are supposed to vary in a chosen form between "nodes" in the boundary, which is also modelled with the same shape functions.

Doing

$$\begin{aligned} \psi &= [N_1(\xi), N_2(\xi), N_3(\xi), \dots] \begin{Bmatrix} \psi_1 \\ \psi_2 \\ \vdots \end{Bmatrix} \\ q &= \partial\psi/\partial\nu = [N_1(\xi), N_2(\xi), N_3(\xi), \dots] \begin{Bmatrix} q_1 \\ q_2 \\ \vdots \end{Bmatrix} \\ ds &= J d\xi. \end{aligned} \quad (5)$$

$N_i$  are shape functions,  $\xi$  natural coordinates in the element and  $J$  the Jacobian to pass from the  $S$  system to the  $\xi$  one.

Equation (4) is discretized to

$$\begin{aligned} c_i \psi_i + \sum_{j=1}^N \left[ \int_{\partial D_j} N_1 \frac{\partial \Delta}{\partial \nu} J d\xi; \int_{\partial D_j} N_2 \frac{\partial \Delta}{\partial \nu} J d\xi; \dots \right] \\ \psi^e = \sum_{j=1}^N \left[ \int_{\partial D_j} N_2 \Delta J d\xi; \int_{\partial D_j} N_2 \Delta J d\xi; \dots \right] q^e. \end{aligned} \quad (6)$$

Subindex  $i$  indicates a boundary node, and  $j$  a boundary element. The integrations are extended to every elemental subdomain  $\partial D_j$ .

The calculation of  $C_i$  is very easy. Assuming a state of constant  $\psi$ ,  $q^e = 0$  and

$$c_i = - \sum_{j=1}^N \left[ \int N_1 \frac{\partial \Delta}{\partial \nu} J d\xi + \int N_2 \frac{\partial \Delta}{\partial \nu} J d\xi + \dots \right]. \quad (7)$$

Repeating eqn (6) for every node a set of algebraic equations of the type

$$H\psi = Gq \quad (8)$$

can be established, where  $H$  and  $G$  are known calculated matrices and  $\psi, q$  are vectors defining the field in the boundary. Nevertheless the size of  $H$  and  $G$  is not the same.

To fix ideas, suppose a plane problem with linear interpolation. There are two shape functions:

$$\begin{aligned} N_1 &= -\frac{1}{2}(\xi - 1) \\ N_2 &= \frac{1}{2}(\xi + 1) \quad -1 < \xi < 1 \\ ds_j &= \frac{L_j}{2} d\xi \end{aligned} \quad (9)$$

and eqn (6) is

$$\begin{aligned} 2c_i \psi_i + \sum_{j=1}^N L_j \left[ \int_{\partial D_j} N_1 \frac{d}{d\xi} \left( \ln \frac{1}{r} \right) d\xi; \right. \\ \left. \int_{\partial D_j} N_2 \frac{d}{d\xi} \left( \ln \frac{1}{r} \right) d\xi \right] \begin{Bmatrix} \psi_j^i \\ \psi_{j+1}^i \end{Bmatrix} = \\ = \sum_{j=1}^N L_j \left[ \int_{\partial D_j} N_1 \ln \frac{1}{r} d\xi; \int_{\partial D_j} N_2 \ln \frac{1}{r} d\xi \right] \begin{Bmatrix} q_j^i \\ q_{j+1}^i \end{Bmatrix} \end{aligned}$$

if element  $j$  runs from  $j$  to  $j+1$  node. Noting that  $q_j^i \neq q_{j+1}^i$  (subindex = node, superindex = element) but  $\psi_j^i = \psi_{j+1}^i$  it's clear that when element  $j$  is viewed from node  $i$ , there are four contributions to matrices  $H, G$ . The first two are produced by the two node potentials  $\psi_j$ ;  $\psi_{j+1}$ ; the other is generated by  $q_j^i$  of node  $j$  and  $q_{j+1}^i$ .  $H$  is then a  $N \times N$  matrix and  $G$  a  $N \times 2N$ , which formation scheme is as follows:

$$\begin{aligned} \begin{bmatrix} i \\ \vdots \\ j \\ \vdots \\ j+1 \\ \vdots \end{bmatrix} \begin{bmatrix} \cdots & \circ & \circ & \cdots \end{bmatrix} \begin{bmatrix} \vdots \\ \vdots \\ \vdots \\ \vdots \end{bmatrix} \begin{bmatrix} \vdots \\ \vdots \\ \vdots \\ \vdots \end{bmatrix} = \\ \begin{matrix} (N \times N) & (N \times 1) \end{matrix} \\ \\ \begin{bmatrix} \vdots \\ \vdots \\ 2j \\ \vdots \\ 2j+1 \\ \vdots \end{bmatrix} \begin{bmatrix} \cdots & \circ & \circ & \cdots \end{bmatrix} \begin{bmatrix} \vdots \\ \vdots \\ \vdots \\ \vdots \end{bmatrix} \begin{bmatrix} \vdots \\ \vdots \\ \vdots \\ \vdots \end{bmatrix} = \\ \begin{matrix} (N \times 2N) & (2N \times 1) \end{matrix} \end{aligned} \quad (11)$$

This fact has not been recognized in previous work [15] because in a Dirichlet condition round a node, one has two unknowns  $q_j^{i-1}, q_j^i$  and only one datum  $\psi_i$ . There are not enough equations to solve the problem. The alternative is to enforce  $q_j^i = q_{j+1}^i$ , reducing  $G$  to a  $N \times N$  matrix but the results are very bad if there is a vertex and, moreover, in the case of a mixed condition one never knows what data to choose. In Ref. [15] this difficulty was circumvented by using two nodes in every sharp corner. The truncation of the corner produces better results in mixed situations but, as would be expected, the Dirichlet condition is not well managed.

Our alternative is directly working in eqn (11). If subindex  $B$  indicates "Before" a node, and  $A$ , "After" a node, there are five possible cases of data round a node

- (1)  $q_B, q_A$  (2)  $q_B, \psi$  (3)  $\psi, q_A$  (4)  $\psi$  sharp corner
- (5)  $\psi$  smooth corner.

This code is the same used in program Betis at the end of the paper. In the three first cases there are two data and only one unknown, so the use of eqn (11) solves the problem. In case 5 the obvious solution is putting  $q_A = q_B$ . Then only case 4 needs a special treatment.

As it is well known

$$\begin{aligned} q_B &= \nabla \psi \cdot \nu_B = q \cos \alpha_B \\ q_A &= \nabla \psi \cdot \nu_A = q \cos \alpha_A \end{aligned} \quad (12)$$

where  $q$  is the gradient modulus and  $\alpha_B, \alpha_A$  the angles between the gradients and boundary normals before and after the node.

If a gradient direction is assumed, the only remaining variable is  $q$  and we have now enough information to use eqn (11). But in a Dirichlet problem one always knows the derivatives along the boundary and then in a sharp corner it is possible to obtain the gradient direction through the two derivatives in it. In the linear hypothesis

$$\begin{aligned}\psi_{x_i} &= \frac{1}{\Delta_i} (y_i - y_{i+1}; y_{i+1} - y_{i-1}; y_{i-1} - y_i) - \begin{Bmatrix} \psi_{i-1} \\ \psi_i \\ \psi_{i+1} \end{Bmatrix} \\ \psi_{y_i} &= \frac{1}{\Delta_i} (x_{i+1} - x_i; x_{i-1} - x_{i+1}; x_i - x_{i-1}) - \begin{Bmatrix} \psi_{i-1} \\ \psi_i \\ \psi_{i+1} \end{Bmatrix} \\ \Delta_i &= (x_{i+1} - x_i)(y_{i-1} - \delta_i) - (x_{i-1} - x_i)(y_{i+1} - y_i) \\ \eta &= \frac{\nabla \psi}{|\nabla \psi|} = \left( \frac{\psi_{1x}}{\sqrt{\psi_{1x}^2 + \psi_{1y}^2}}; \frac{\psi_{1y}}{\sqrt{\psi_{1x}^2 + \psi_{1y}^2}} \right)\end{aligned}$$

where  $(i-1), (i+1)$  are the nodes before and after the  $i$  node under study.

Using  $\eta$  it is possible to compute  $\alpha_B, \alpha_A$  and after solving the system of  $N$  equations, a back substitution in (12) will show the desired values  $q_B, q_A$ .

The guess can be refined using an iterative process, but the run of several examples has shown very good results with this simple approach.

In the case of parabolic of higher order elements a tangent plane can always be guessed, the only change necessary being in eqn (13).

Another improvement of program Betis is the automatic fashion in which the system (11) is substituted by a

$$KX = F \quad (14)$$

directly prepared to be acted by a standard routine solver.

To avoid the storage of matrices  $H, G$  the program has been prepared to form the product of every calculated coefficient by its corresponding data and to put it in vector  $F$ . If the code of the point under study indicates the presence of an unknown, this is relocated in the adequate row of  $X$  and so is its correlated coefficient in matrix  $K$ . Repeating this process for every node one can directly form the (eqn 14) using storage for an  $N \times N$  matrix and two  $N \times 1$  vectors.

Careful programming has permitted also an optimal use of the in core memory of the Hewlett. Packard 21-MX in which the following examples have been run.

### 3. B.I.E.M. TECHNIQUE IN ELASTICITY THEORY

The reciprocity theorem in elasticity is the well known Betti's theorem

$$\int_{\partial D} T^n \cdot u^* ds + \int_D x \cdot u^* dv = \int_{\partial D} T^n \cdot u ds + \int_D x^* u dv \quad (15)$$

which in absence of volume forces is reduced to

$$\int_{\partial D} T^n \cdot u^* ds = \int_{\partial D} T^{n*} \cdot u ds + \int_D x^* u dv. \quad (16)$$

The second feature of BIEM is the fundamental solution than can be obtained with a Kelvin solution

$$\begin{aligned}u_k^* &= U_{ki} e_i \\ T_k^{n*} &= T_{ki} e_i\end{aligned} \quad (17)$$

where  $e_i$  are directions of actuation of unitarian loads. In two dimensions

$$\begin{aligned}u_{ki} &= \frac{1}{8\pi\mu(1-\nu)} \left[ (3-4\nu) \ln \frac{1}{r} \delta_{ki} + r_{1k} r_{.i} \right] \\ T_{ki} &= -\frac{1}{4\pi(1-\nu)} \frac{1}{r} \left\{ \frac{dr}{dn} [(1-2\nu)\delta_{ki} + 2r_{.i} r_{2k}] \right. \\ &\quad \left. + (1-2\nu)[n_i r_{.k} - n_k r_{.i}] \right\}.\end{aligned} \quad (18)$$

Where  $T$  are the tractions,  $u$  the displacement vector,  $r$  the distance between the point under study and that in which the unit load is placed,  $\mu = E/2(1+\nu)$ ,  $E$  Young modulus and  $\nu$  Poisson coefficient.

Substituting the Kelvin solution in (16) one gets

$$u_i(p) + \int_{\partial D} T_{ki}(p, q) u_k(q) ds = \int_{\partial D} u_{ki}(p, q) T_k^n(q) ds \quad (19)$$

which is known as Somigliana's identity.

Up from here the process is absolutely the same used in the previous paragraph.

Establishing the equation in the contour,

$$c(p)u(p) + \int_{\partial D} T \cdot u ds = \int_{\partial D} U \cdot T^n ds. \quad (20)$$

Doing the linear discretization

$$\begin{aligned}u &= \begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} N_1 & N_2 & 0 & 0 \\ 0 & 0 & N_1 & N_2 \end{bmatrix} \begin{Bmatrix} \bar{X}_1^e \\ \bar{X}_2^e \\ \bar{Y}_1^e \\ \bar{Y}_2^e \end{Bmatrix} \\ ds &= \frac{L^e}{2} d\xi \\ N_1 &= -\frac{1}{2}(\xi - 1) \\ N_2 &= +\frac{1}{2}(\xi + 1)\end{aligned} \quad (21)$$

with the usual notations, eqn (20) can be written for every node  $i$

$$\begin{aligned}2c_i \begin{Bmatrix} u_i \\ v_i \end{Bmatrix} + \sum_{j=1}^N L_j \begin{bmatrix} \int_{\partial D_j} N_1 T_{11} d\xi & \int_{\partial D_j} N_2 T_{11} d\xi & \int_{\partial D_j} N_1 T_{12} d\xi & \int_{\partial D_j} N_2 T_{12} d\xi \\ \int_{\partial D_j} N_1 T_{21} d\xi & \int_{\partial D_j} N_2 T_{21} d\xi & \int_{\partial D_j} N_1 T_{22} d\xi & \int_{\partial D_j} N_2 T_{22} d\xi \end{bmatrix} \begin{Bmatrix} u_j^i \\ v_j^i \\ u_{j+1}^i \\ v_{j+1}^i \end{Bmatrix} = \\ = \sum_{j=1}^N L_j \begin{bmatrix} \int_{\partial D_j} N_1 U_{11} d\xi & \int_{\partial D_j} N_2 U_{11} d\xi & \int_{\partial D_j} N_1 U_{12} d\xi & \int_{\partial D_j} N_2 U_{12} d\xi \\ \int_{\partial D_j} N_1 U_{21} d\xi & \int_{\partial D_j} N_2 U_{21} d\xi & \int_{\partial D_j} N_1 U_{22} d\xi & \int_{\partial D_j} N_2 U_{22} d\xi \end{bmatrix} \begin{Bmatrix} \bar{X}_j^i \\ \bar{X}_{j+1}^i \\ \bar{Y}_j^i \\ \bar{Y}_{j+1}^i \end{Bmatrix}.\end{aligned} \quad (22)$$

As usual a subindex means "node" and a superindex "element".  $(\bar{X}, \bar{Y})$  are the coordinates components of boundary tractions and  $T_{kl}, U_{ke}$  are the components of the matrices  $T$  and  $U$  of the fundamental solution used.

In this point a transformation will be made: displacements will remain in global coordinates but tractions will be expressed as normal and tangential components.

$$\begin{Bmatrix} \sigma \\ \tau \end{Bmatrix}^e = \begin{pmatrix} \cos \alpha_e & \sin \alpha_e \\ -\sin \alpha_e & \cos \alpha_e \end{pmatrix} \begin{Bmatrix} \bar{X} \\ \bar{Y} \end{Bmatrix} \quad (23)$$

where  $\alpha_e$  is the  $x$ -axis, normal to element) angle.

Equation (22) will now be

$$c_i \begin{Bmatrix} u_i \\ v_i \end{Bmatrix} + \sum_{j=1}^N \begin{bmatrix} A_{11}^j & A_{12}^j & A_{13}^j & A_{14}^j \\ A_{21}^j & A_{22}^j & A_{23}^j & A_{24}^j \end{bmatrix} \begin{Bmatrix} u_{j+1}^j \\ v_{j+1}^j \end{Bmatrix} = \sum_{j=1}^N \begin{bmatrix} B_{11}^j & B_{12}^j & B_{13}^j & B_{14}^j \\ B_{21}^j & B_{22}^j & B_{23}^j & B_{24}^j \end{bmatrix} \begin{Bmatrix} \sigma_{j+1}^j \\ \tau_{j+1}^j \end{Bmatrix} \quad (24)$$

Varying  $i$  one can obtain a system of  $2N$  equations

$$AU = B\sigma. \quad (25)$$

The scheme of contribution of each element is  $f$  or matrix  $H$

$$\begin{array}{c} \begin{matrix} (i) \\ (N+i) \end{matrix} \begin{bmatrix} \cdots & A_{11}^j & A_{12}^j & \cdots & A_{13}^j & A_{14}^j & \cdots \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \\ \cdots & A_{21}^j & A_{22}^j & \cdots & A_{23}^j & A_{24}^j & \cdots \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \end{bmatrix} \begin{Bmatrix} u_j \\ u_{j+1} \\ \vdots \\ v_j \\ v_{j+1} \end{Bmatrix} = \end{array}$$

$(2N \times 2N)$   $(2N+1)$

$$= \begin{array}{c} \begin{matrix} (i) \\ (N+i) \end{matrix} \begin{bmatrix} \cdots & B_{11}^j & B_{12}^j & \cdots & B_{13}^j & B_{14}^j & \cdots \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \\ \cdots & B_{21}^j & B_{22}^j & \cdots & B_{23}^j & B_{24}^j & \cdots \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \end{bmatrix} \begin{Bmatrix} \sigma_j^j \\ \sigma_{j+1}^j \\ \vdots \\ \tau_j^j \\ \tau_{j+1}^j \end{Bmatrix}$$

As in the previous paragraph some authors reduce the size of  $B$  equating the tractions round each node[10]. Nevertheless one has enough data use eqn (25) except when the conditions are in displacements, because then there are four unknowns:  $\sigma_B, \tau_B, \sigma_A, \tau_A$  and only two equations to be used.

The reduction of 4 unknowns to two follows the same pattern as in potential theory, the dummy variables being now the principal stresses in the corner.

If one defines

$$\begin{aligned} \cos \vartheta &= \underline{n}_I \cdot \underline{n} \\ \sin \vartheta &= \underline{n}_{II} \cdot \underline{n} \end{aligned} \quad (26)$$

where  $\underline{n}$  is the element normal and  $\underline{n}_I, \underline{n}_{II}$  the principal directions of stresses, it can be easily shown

$$\begin{Bmatrix} \sigma \\ \tau \end{Bmatrix}^n = \begin{bmatrix} \cos^2 \alpha & \sin^2 \alpha \\ -\sin \alpha \cos \alpha & \sin \alpha \cos \alpha \end{bmatrix} \begin{Bmatrix} \sigma_I \\ \sigma_{II} \end{Bmatrix} \quad (27)$$

being  $\sigma_I, \sigma_{II}$  the principal stresses.

Transformation(27) is followed in corners where only displacements are known.

To obtain  $n_I$  and  $n_{II}$ ,  $u$  and  $v$ , are assumed to behave as plane functions near the corner. Then one can define

$$\begin{aligned} \epsilon_x &= u_{,x} \\ \epsilon_y &= v_{,y} \\ \epsilon_{xy} &= u_{,y} + v_{,x} \end{aligned} \quad (28)$$

in function of the  $(u_{i-1}; u_i; u_{i+1}), (v_{i-1}; v_i; v_{i+1})$  values in nodes  $(i-1); (i+1)$  before and after that in study.

Eigenvalues of  $\epsilon$  tensor

$$\epsilon_{I,II} = \frac{\epsilon_x + \epsilon_y}{2} \pm \frac{1}{2} \sqrt{((\epsilon_x - \epsilon_y)^2 + 4\gamma_{xy}^2)} \quad (29)$$

and eigenvectors

$$\underline{n}_{I,II} = \frac{1}{\sqrt{((\epsilon_y - \epsilon_x)^2 + \gamma_{xy}^2)}} \begin{Bmatrix} \epsilon_y - \epsilon_x \\ -\gamma_{xy} \end{Bmatrix} \quad (30)$$

will be subsequently obtained.

Equation (30) are the normals needed in eqn (26) because in isotropic elasticity principal axis are common to stress and strain tensors.

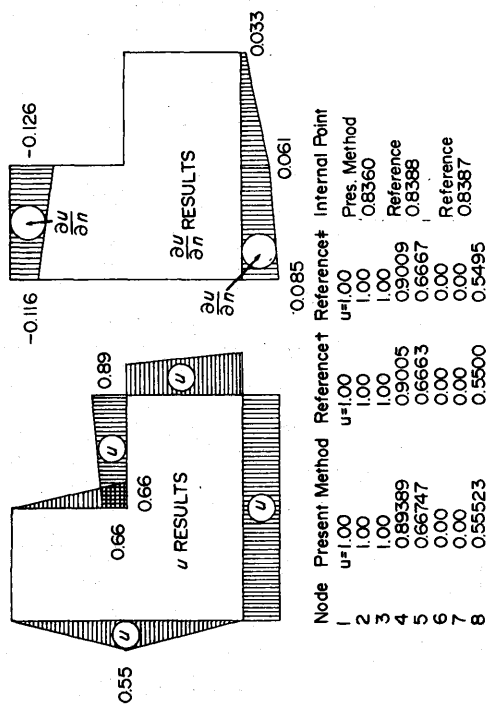
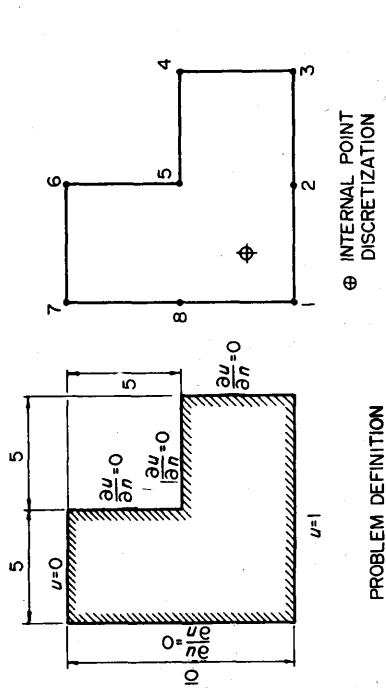


Fig. 1.

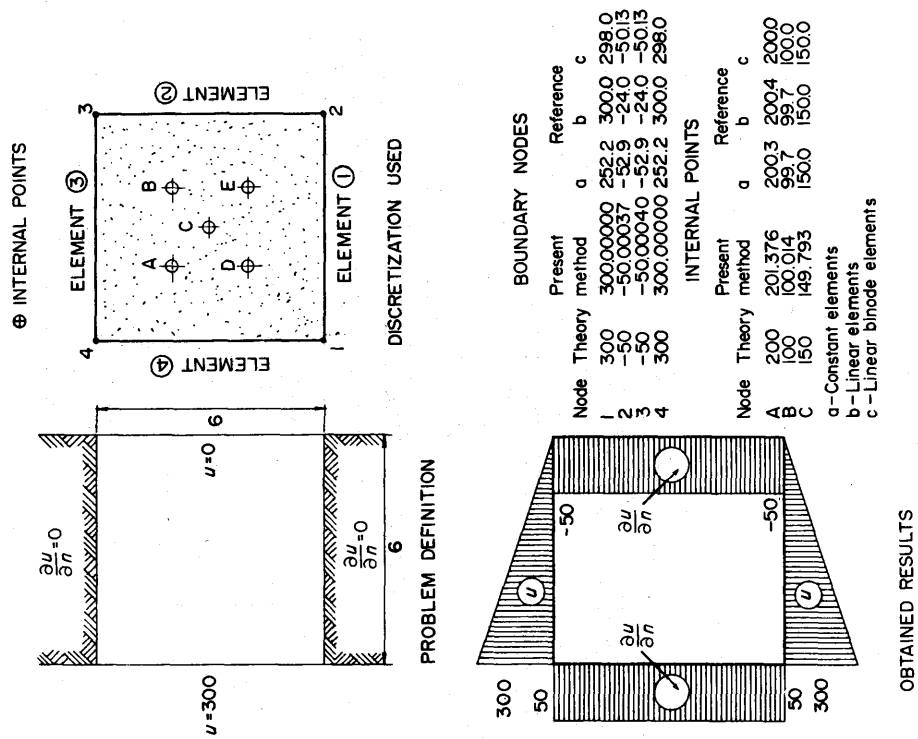


Fig. 2.

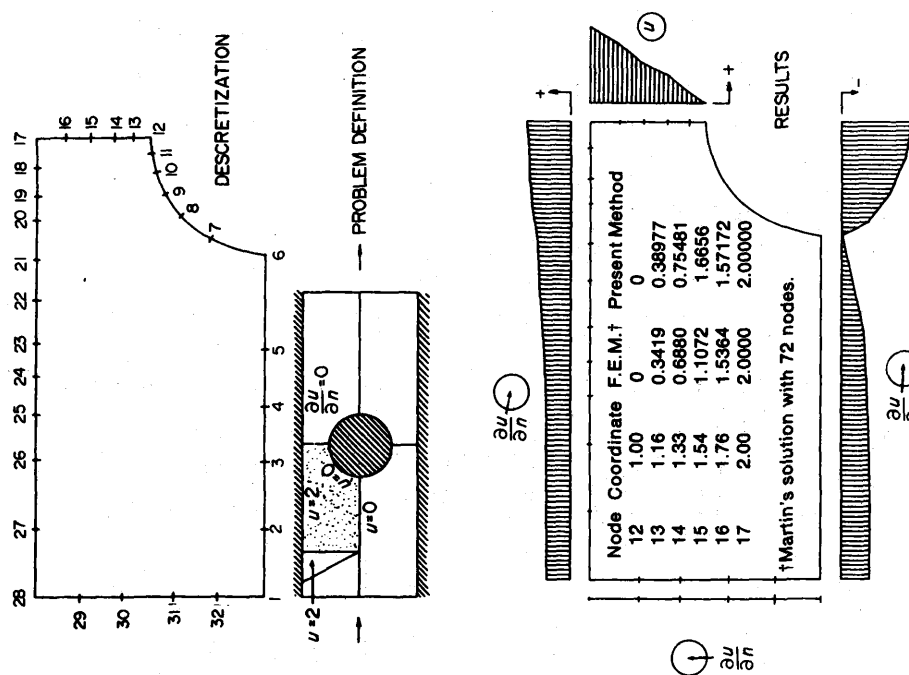


Fig. 3.

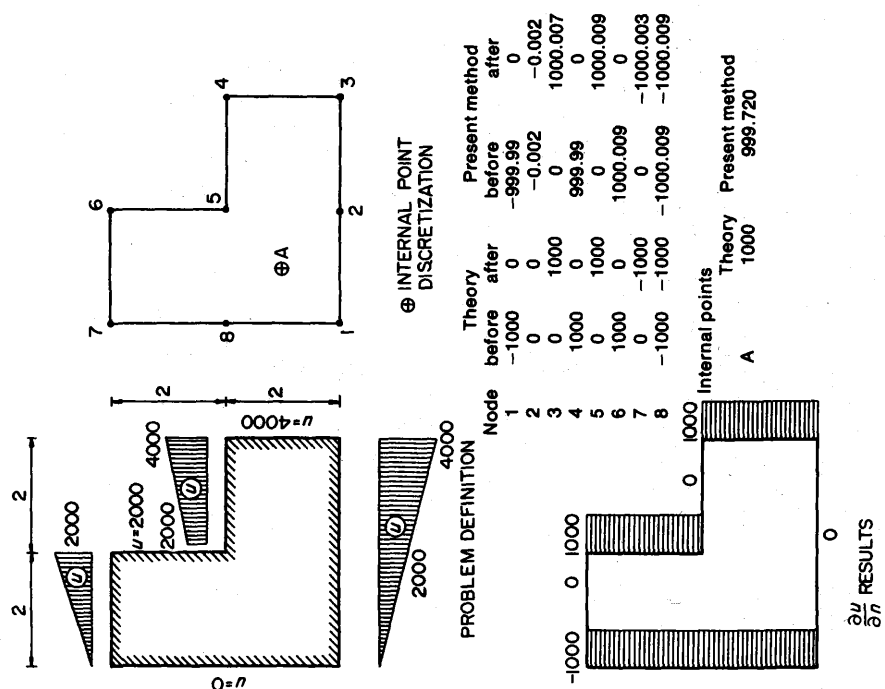
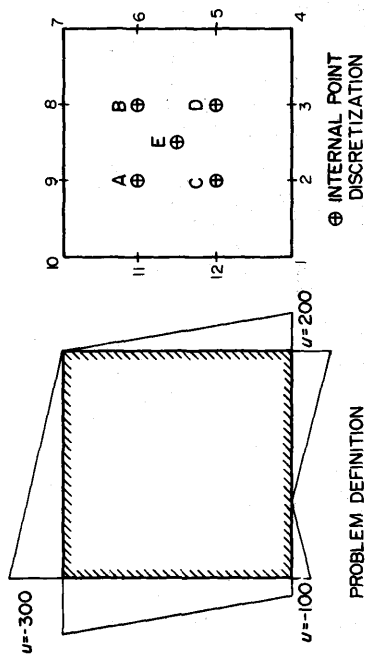


Fig. 4.



Node	Theory		Present method	
	before	after	before	after
1	-50.00	33.33	-50.0000	33.3336
2	33.33	33.33	33.3336	33.3331
3	33.33	33.33	33.3331	33.3331
4	33.33	50.00	33.3334	50.0001
5	50.00	50.00	49.9998	49.9998
6	50.00	50.00	50.0003	50.0003
7	50.00	-33.33	-49.9996	-33.3331
8	-33.33	-33.33	-33.3336	-33.3336
9	-33.33	-33.33	-33.3331	-33.3331
10	-33.33	-50.00	-33.3333	-50.0000
11	-50.00	-50.00	-49.9999	-49.9999
12	-50.00	-50.00	-50.0001	-50.0001

Interior points	
Theory	Present method
-133.3333	-133.334
-33.3333	-33.333
-66.6666	-66.667
33.3333	33.334
-50.0000	-50.000

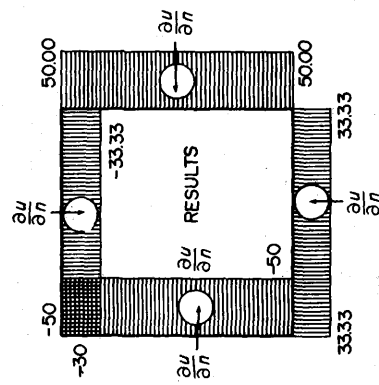


Fig. 5.

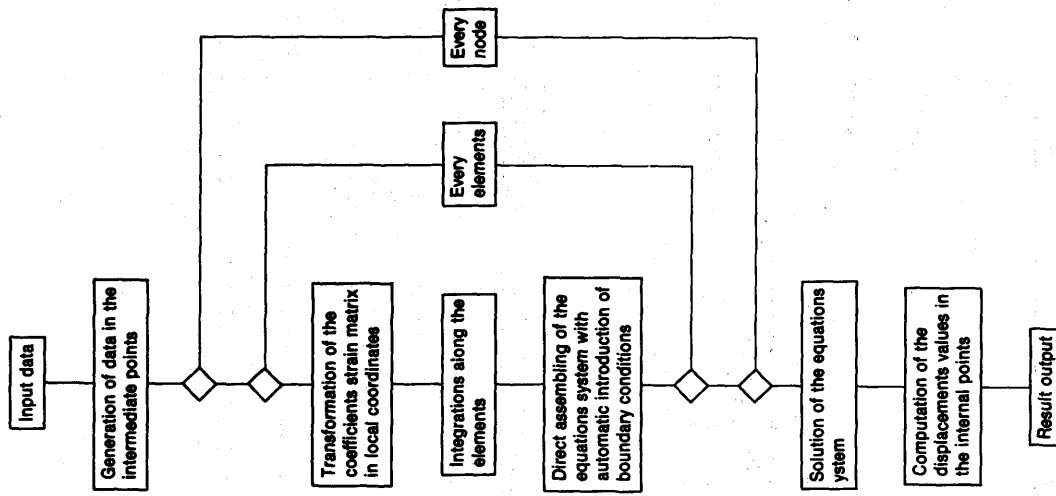


Fig. 6. Flow chart program SERBA.

As in potential theory this guess have proved to be very efficient.

The program Serba, whose flow-chart is presented in Fig. 6, uses this improvement. In it eqn (24) is not managed. Instead a system

$$KX = F \quad (31)$$

is directly formed following the same philosophy than in potential theory.

As previously noted this direct procedure of assemblage is very well suited to small computers.

#### 4. FUTURE TRENDS IN B.I.E. METHODS

Several interesting advantages can be attributed to B.I.E.M. over F.E. techniques.

(1) The ease of treatment of infinite domains which arises in soil-mechanics or hydraulics problems.

This is specially appreciated in dynamic soil-structure or fluid-structure problems, when F.E.M. needs a new tool: the absorbing boundary.

(2) If F.E.M. is established on displacements, the stresses are worse modelled than those. In B.I.E.M. the internal results have the same degree of accuracy. This is very interesting when one is studying situations in which concentration of stresses are present.

(3) The reduction in dimension (3D→2D: 2D→1D) of the domain to be discretized also reduces the size of the problem to be handled. This is important because it means a reduction in the input data to be prepared, the interpretation of results and the in-core computer occupation.

All this features will push the applications of B.I.E.M. to dynamics, plasticity, termoelasticity, viscoplasticity, anisotropic problems, etc. Several papers have already been published and it is clear that, in the future, more work will be dedicated to these fields.

In the other hand, in spite of its virtues, B.I.E.M. have gross problems in front of its computer implementation.

The matrices to be used are fully populated and non-symmetric. Another future trend will be the implementation of good numerical techniques to treat this situation and, also, the establishment of new procedures (substructuring, subregionalization, etc.)

Finally more effort will be dedicated to the coupling with F.E. in order to obtain the best possible analysis capability.

#### 5. CONCLUSIONS

(i) A comprehensive treatment of potential and elasticity problems, in the light of B.I.E.M. technique, has been outlined.

(ii) Two new ideas described are the treatment of sharp-corner boundaries and the direct assembling of the equation system. Both ideas can be easily extended to all classes of isoparametric elements and dimension problems.

(iii) Two simple computer programs have been developed to show the ease implementation of the procedure.

(iv) The method has been applied to a number of problems showing its good work quality.

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#### APPENDIX

##### HEAT FLUX IN A PLATE WITH FOUR ELEMENTS

NODE	COORD X	COORD Y	CODE	FUNCTION	FLUX BEFORE	FLUX AFTER
1	.0000	.0000	2	300.00000	.00000	.00000
2	6.0000	.0000	3	.00000	.00000	.00000
3	6.0000	6.0000	2	.00000	.00000	.00000
4	.0000	6.0000	3	300.00000	.00000	.00000

##### HEAT FLUX IN A PLATE WITH FOUR ELEMENTS

NODE	FUNCTION	FLUX BEFORE	FLUX AFTER
1	300.00000	50.00040	.00000
2	.00000	.00000	-50.00037
3	.00000	-50.00040	.00000
4	300.00000	.00000	50.00031



## INTERNAL POINTS

COORD X	COORD Y	FUNCTION
2.000	2.000	201.376
2.000	4.000	201.376
3.000	3.000	149.793
4.000	2.000	100.014
4.000	4.000	100.014

NASAS T=00003 IS ON CR00010 USING 00010 BLKS R=0000

001 FTM4,L

002 SUBROUTINE HGNM(XP,YP,X1,Y1,X2,Y2,A1,A2,B1,B2)

003 C

004 C THIS SUBROUTINE COMPUTE NUMERICALLY THE INTEGRALS ALONGS AN ELEMENT

005 C WHICH DOES NOT INCLUDE THE NODE UNDER CONSIDERATION

006 C

007 DIMENSION X1(4),OMEG(4)

008 DATA X1/0.86113631,-0.86113631,0.33998104,-0.33998104/,

009 1 OMEG/0.34785485,0.34785485,0.65214515,0.65214515/

010 AX=(X2-X1)/2

011 AY=(Y2-Y1)/2

012 BX=(X2+X1)/2

013 BY=(Y2+Y1)/2

014 IF(AX)10,20,10

015 10 DIS=ABS((AY\*XP/AX-YP+Y1-AY\*X1/AX)/SQRT((AY/AX)\*\*2+1))

016 GO TO 30

017 20 DIS=ABS(XP-X1)

018 30 SIG=(X1-XP)\*(Y2-YP)-(X2-XP)\*(Y1-YP)

019 IF(SIG.LT.0) DIS=-DIS

020 A1=0.

021 A2=0.

022 B1=0.

023 B2=0.

024 DO 40 I=1,4

025 XC=AX\*X1(I)+BX

026 YC=AY\*X1(I)+BY

027 R=SQRT((XP-XC)\*\*2+(YP-YC)\*\*2)

028 H=DIS\*OMEG(I)\*SQRT(AX\*\*2+AY\*\*2)/R\*\*2

029 G=ALOG(1/R)\*OMEG(I)\*SQRT(AX\*\*2+AY\*\*2)

030 A1=A1+(X1(I)-1)\*H/2

031 A2=A2-(X1(I)+1)\*H/2

032 B1=B1-(X1(I)-1)\*G/2

033 40 B2=B2+(X1(I)+1)\*G/2

034 RETURN

035 END

036 SUBROUTINE HGANA(X1,Y1,X2,Y2,A1,A2,B1,B2)

037 C

038 C THIS SUBROUTINE COMPUTE ANALITICALLY THE INTEGRALS ALONG AN ELEMENT

039 C INCLUDING THE NODE UNDER CONSIDERATION

040 C

041 A1=0.

042 A2=0.

043 DIS=SQRT((X2-X1)\*\*2+(Y2-Y1)\*\*2)

044 B1=DIS\*(1.5-ALOG(DIS))/2

045 B2=DIS\*(0.5-ALOG(DIS))/2

046 RETURN

047 END

048 SUBROUTINE SOLUG(A,N,C,IPIVO,NPMA)

049 C

050 C THIS SUBROUTINE SOLVE LINEAR SYSTEMS OF EQUATIONS BY THE GAUSS

051 C ELIMINATION METHOD

052 C

053 DIMENSION A(NPMA,NPMA),IPIVO(NPMA),C(NPMA)

054 IMP=6

055 DO 9 M=1,N

056 9 IPIVO(M)=0

057 DO 55 J=1,N

058 DMAX=0.0

059 DO 200 K=1,N

060 IF(IPIVO(K)-1)31,200,31

061 31 IF(ABS(DMAX)-ABS(A(K,K)))32,200,200

062 32 DMAX=A(K,K)

063 IR=K

064 200 CONTINUE

065 IF(ABS(DMAX)-2.0E-8)600,600,40

066 600 WRITE(IMP,99)

067 99 FORMAT('THE MATRIX IS SINGULAR')

068 40 IPIVO(IR)=IPIVO(IR)+1

069 A(IR,IR)=1.0

070 DO 41 I=1,N

071 41 A(IR,I)=A(IR,I)/DMAX

072 C(IR)=C(IR)/DMAX

073 DO 55 K=1,N

074 IF(K-IR)42,55,42

075 42 R=A(K,IR)

076 A(K,IR)=0.0

077 DO 45 L=1,N

078 45 A(K,L)=A(K,L)-A(IR,L)\*R

079 C(K)=C(K)-C(IR)\*R

080 55 CONTINUE

081 RETURN

082 END

083 \$

```

NASAP T=00003 IS ON CR00010 USING 00024 BLKS R=0000
001 FTN4,L
002 PROGRAM BETIS(3)
003 C
004 C THIS IS A THREE SEGMENTS PROGRAM
005 C SEGMENT BETI1: GENERATION INPUT DATA
006 C SEGMENT BETI2: FORMULATION AND SOLUTION OF THE SYSTEM OF EQUATIONS
007 C SEGMENT BETI3: INTERNAL POINTS COMPUTATIONS
008 C
009 DIMENSION NOM(3)
010 COMMON ACOM(2410)
011 DATA NOM/2HBE,2HT1,2H1 /
012 CALL EXEC (8,NOM)
013 END
014 C
015 C FIRST SEGMENT STARTING
016 C
017 PROGRAM BETI1(5)
018 DIMENSION X(47),Y(47),NCOD(47),IENC(25),NOM(3),FI(47),DFI(47,2)
019 DIMENSION XINT(46),YINT(46)
020 COMMON IMP,LEC,NP,NP1,NSUP,IENC,NPMAX,X,Y,NCOD,FI,DFI,XINT,YINT
021 DATA NOM/2HBE,2HT1,2H2 /
022 C
023 C ASSIGN DATA SET NUMBERS FOR INPUT, LEC, AND OUTPUT, IMP
024 C
025 LEC=7
026 IMP=6
027 NPMAX=46
028 C
029 C READ BASIC PARAMETERS
030 C NP=NUMBER OF BOUNDARY NODES
031 C NPY=NUMBER OF INTERNAL POINTS WHERE THE FUNCTION WILL BE CALCULATED
032 C IENC=PROBLEM TITLE
033 C
034 READ(LEC,1000) NP,NP1,IENC
035 DO 7 I=1,NP
036 FI(I)=0.
037 DFI(I,1)=0.
038 7 DFI(I,2)=0.
039 1000 FORMAT(2I5,10X,25A2)
040 C
041 C READ BASIC PARAMETERS
042 C XINT,YINT= INTERNAL POINTS COORDINATES
043 C
044 READ(LEC,1005)(XINT(I),YINT(I),I=1,NP1)
045 1005 FORMAT(2F10.0)
046 L=0
047 C
048 C READ BASIC PARAMETERS
049 C X,Y=BOUNDARY POINTS COORDINATES
050 C
051 5 READ(LEC,1010)I,X(I),Y(I)
052 1010 FORMAT(I5,2F10.0)
053 IF(L)15,10,15
054 15 NINT=I-L
055 AX=(X(I)-X(L))/NINT
056 AY=(Y(I)-Y(L))/NINT
057 10 L=L+1
058 IF(I-L)500,25,20
059 20 X(L)=X(L-1)+AX
060 Y(L)=Y(L-1)+AY
061 GO TO 10
062 25 IF(NP-I)24,26,5
063 26 X(NP+1)=X(I)
064 Y(NP+1)=Y(I)
065 24 L=0
066 C
067 C READ BASIC PARAMETERS
068 C NCOD(I)=CODE OF THE NODE I ACCORDING TO THE NEIGHBOUR CONDITIONS
069 C CODE 1 = THE VALUES OF THE FUNCTION DERIVATIVE BEFORE AND
070 C AFTER OF THE NODE ARE KNOWN
071 C CODE 2 = THE VALUE OF THE FUNCTION BEFORE AND THE VALUE OF THE
072 C FUNCTION DERIVATIVE AFTER ARE KNOWN
073 C CODE 3 = THE VALUE OF THE FUNCTION DERIVATIVE BEFORE AND THE
074 C VALUE OF THE FUNCTION AFTER ARE KNOWN
075 C CODE 4 = THE VALUES OF THE FUNCTION BEFORE AND AFTER A SHARP
076 C CORNER ARE KNOWN
077 C CODE 5 = US 4 BUT IN CONTINUOUS BOUNDARY
078 C FI(I) = THE VALUE OF THE FUNCTION IN THE NODE I, IF IS KNOWN
079 C DFI(I,1),DFI(I,2) = THE VALUES OF THE DERIVATIVE FUNCTIONS BEFORE
080 C AND AFTER IF IS KNOWN
081 C
082 30 READ(LEC,1015)I,NCOD(I),FI(I),DFI(I,1),DFI(I,2)
083 1015 FORMAT(2I5,3F10.0)
084 IF(I-L)502,35,40
085 40 NINT=I-L
086 AX=(FI(I)-FI(L))/NINT
087 AY=(DFI(I,1)-DFI(L,1))/NINT
088 35 L=L+1
089 IF(I-L)502,55,42
090 42 IF(NCOD(I).LE.3.AND.NCOD(I).NE.2) GO TO 41
091 NCOD(L)=5
092 FI(L)=FI(L-1)+AX

```

```

093      GO TO 35
094      41 NCOD(L)=1
095      DF I(L,1)=DF I(L-1,2)+AY
096      DF I(L,2)=DF I(L,1)
097      GO TO 35
098      55 IF(NP-1)54,56,30
099      56 F I(NP+1)=F I(1)
100      DF I(NP+1,1)=DF I(1,1)
101      DF I(NP+1,2)=DF I(1,2)
102      54 WRITE(IMP,2000)IENC
103      2000 FORMAT(1H1,10X,25A2/3X,"NODE",12X,"COORD X",9X,"COORD Y",9X," CODE "
104      1,10X," FUNCTION",6X," FLUX BEFORE",5X," FLUX AFTER")
105      WRITE(IMP,1020)(1,X(1),Y(1),NCOD(1),F I(1),(DF I(1,K),K=1,2),I=1,NP)
106      1020 FORMAT(2X,15,3X,2F15.4,15,1X,3F20.5)
107      500 CONTINUE
108      502 CONTINUE
109      CALL EXEC(8,NOM)
110      END
111 C
112 C SECOND SEGMENT STARTING
113 C
114      PROGRAM BET I2(5)
115      DIMENSION X(47),Y(47),NCOD(47),IENC(25),NOM(3),F I(47),DF I(47,2)
116      DIMENSION XINT(46),YINT(46)
117      DIMENSION COEF(46,46),CARGA(46),A(2),B(2),IPIVO(46)
118      COMMON IMP,LEC,NP,NF,I,NSUP,IENC,NPMAX,X,Y,NCOD,F I,DF I,XINT,YINT
119      DATA NOM/2HBE,2HT1,2H3 /
120      DIS(X1,Y1,X2,Y2)=SQRT((X1-X2)**2+(Y1-Y2)**2)
121      DO 100 NODO=1,NP
122      DO 10 I=1,NF
123      10 COEF(NODO,I)=0.
124      CD IAG=0.
125      CARGA(NODO)=0.
126 C
127 C IF THE ELEMENTS INCLUDE THE NODE UNDER CONSIDERATION,THE INTEGRATION
128 C ON THE BOUNDARY ELEMENTS ARE COMPUTED ANALITICALLY (SUBROUTINE HGANA),
129 C IF NOT THOSE ARE NUMERICALLY DONE (SUBROUTINE HGNUM)
130 C
131      DO 90 IELEM=1,NP
132      IF(IELEM.EQ.NODO)GO TO 11
133      IF((IELEM+1).EQ.NODO)GO TO 12
134      IF(IELEM.EQ.NP.AND.NODO.EQ.1)GO TO 12
135      CALL HGNUM(X(NODO),Y(NODO),X(IELEM),Y(IELEM),X(IELEM+1),
136      Y(IELEM+1),A(1),A(2),B(1),B(2))
137      GO TO 15
138      11 CALL HGANA(X(NODO),Y(NODO),X(IELEM+1),Y(IELEM+1),A(1),A(2),B(1),
139      B(2))
140      GO TO 15
141      12 CALL HGANA(X(IELEM),Y(IELEM),X(NODO),Y(NODO),A(1),A(2),B(1),B(2))
142      AX=B(1)
143      B(1)=B(2)
144      B(2)=AX
145      15 DIAG=CD IAG+A(1)+A(2)
146      DO 80 K=1,2
147      KN=IELEM+K-1
148      IF(KN.EQ.NP+1)KN=1
149 C
150 C THE INTEGRATIONS RESULTS ARE PLACED IN THE COEFFICIENTS MATRIX OR IN
151 C THE LOAD VECTOR ACCORDING TO THE CODE OF NODES
152 C
153      GO TO(1,2,3,4,5),NCOD(KN)
154      1 CARGA(NODO)=CARGA(NODO)+B(K)*DF I(KN,3-K)
155      COEF(NODO,KN)=COEF(NODO,KN)+A(K)
156      GO TO 80
157      2 IF(K.EQ.2)GO TO 31
158      21 CARGA(NODO)=CARGA(NODO)+B(K)*DF I(KN,3-K)-A(K)*F I(KN)
159      GO TO 80
160      3 IF(K.EQ.2)GO TO 21
161      31 CARGA(NODO)=CARGA(NODO)-A(K)*F I(KN)
162      COEF(NODO,KN)=COEF(NODO,KN)-B(K)
163      GO TO 80
164      4 DEN=DIS(X(IELEM),Y(IELEM),X(IELEM+1),Y(IELEM+1))
165      XNUE=(Y(IELEM+1)-Y(IELEM))/DEN
166      YNUE=(X(IELEM)-X(IELEM+1))/DEN
167      KNMA=KN+1
168      KNME=KN-1
169      IF(KN.EQ.1)KNME=NP
170      IF(KN.EQ.NP)KNMA=1
171      DEN=(X(KNMA)-X(KN))* (Y(KNME)-Y(KN))-(X(KNME)-X(KN))* (Y(KNMA)-Y(KN))
172      1)
173 C
174 C COMPUTE GRADIENT COMPONENTS IN THE NODE UNDER STUDY
175 C
176      DFX=((Y(KN)-Y(KNMA))* F I(KNME)+((Y(KNMA)-Y(KNME))* F I(KN)+(Y(KNME)-Y(
177      KN)) F I(KNMA))/DEN
178      DFY=((X(KNMA)-X(KN))* F I(KNME)+(X(KNME)-X(KNMA))* F I(KN)+(X(KN)-X(KN
179      ME)) F I(KNMA))/DEN
180      COSE=0.7071
181      IF(DFX.NE.0.) GO TO 43
182      IF(DFY.EQ.0.) GO TO 44
183      43 COSE=(DFX*XNUE+DFY*YNUE)/SQRT(DFX**2+DFY**2)
184      44 KK=3-K
185      41 DF I(KN,KK)=COSE

```

```

186      CCEF(NODO,KN)=COEF(NODO,KN)-B(K)*COSE
187      CARGA(NODO)=CARGA(NODO)-A(K)*F1(KN)
188      GO TO 80
189      5 COSE=1
190      GO TO 41
191      80 CONTINUE
192      90 CONTINUE
193 C
194 C MODIFY THE ELEMENTS OF THE PRINCIPAL DIAGONAL IN THE COEFFICIENTS
195 C MATRIX OF THE FUNCTIONS
196 C
197      IF(NCOD(NODC)-1)91,92,91
198      91 CARGA(NODO)=CARGA(NODO)+CDIAG*F1(NODO)
199      GO TO 100
200      92 COEF(NODO,NODC)=COEF(NODO,NODO)-CDIAG
201      100 CONTINUE
202 C
203 C SOLUTION OF THE SYSTEM OF EQUATIONS
204 C
205      CALL SOLUG(COEF,NP,CARGA,IPIVO,NPMAX)
206      DO 200 NODO=1,NP
207      NC=NCOD(NODO)
208 C
209 C ASSEMBLE AND MODIFY THE RESULTANTS VALUES OF THE SYSTEM OF EQUATIONS
210 C
211      GO TO(201,202,202,204,205),NC
212      201 F1(NODO)=CARGA(NODO)
213      GO TO 200
214      202 NC=NC-1
215      DF1(NODO,NC)=CARGA(NODO)
216      GO TO 200
217      204 DO 210 I=1,2
218 C      IF(ABS(DF1(NODO,I)).LE=1.E-18) GO TO 210
219      DF1(NODO,I)=CARGA(NODO)*DF1(NODO,I)
220      210 CONTINUE
221      GO TO 200
222      205 DO 220 I=1,2
223      220 DF1(NODO,I)=CARGA(NODO)
224      200 CONTINUE
225      CALL EXEC(8,NOM)
226      END
227 C
228 C THIRD SEGMENT STARTING
229 C
230      PROGRAM BET13(5)
231      DIMENSION X(47),Y(47),NCOD(47),IENC(25),NOM(3),F1(47),DF1(47,2)
232      DIMENSION XINT(46),YINT(46)
233      DIMENSION FINT(46),A(2),B(2)
234      COMMON IMP,LEC,NP,NP1,NSUP,IENC,NPMAX,X,Y,NCOD,F1,DF1,XINT,YINT
235      DO 100 I=1,NP1
236      FINT(I)=0.
237      DO 50 J=1,NP
238 C
239 C COMPUTE NUMERICALLY THE INTEGRATION VALUES ALONG BOUNDARY ELEMENTS
240 C
241      CALL HGNUM(XINT(I),YINT(I),X(J),Y(J),X(J+1),Y(J+1),A(1),A(2),B(1),
242      1B(2))
243      JJ=J+1
244      IF(J.EQ.NP)JJ=1
245 C
246 C COMPUTE DE FUNCTION VALUE IN THE INTERNAL POINTS
247 C
248      50 FINT(I)=FINT(I)-A(1)*F1(J)-A(2)*F1(JJ)+B(1)*DF1(J,2)+B(2)
249      1DF1(JJ,1)
250      100 FINT(I)=FINT(I)/(2*3.1415926)
251      WRITE(IMP,2100)IENC
252      2100 FORMAT(1H1,10X,25A2/10X,50(" ")/5X," BOUNDARY POINTS " /6X," NODE"
253      1,11X," FUNCTION",6X," FLUX BEFORE",5X," FLUX AFTER")
254      WRITE(IMP,2101)(I,F1(I),(DF1(I,J),J=1,2),I=1,NP)
255      2101 FORMAT(5X,15,3F20.5)
256      WRITE(IMP,2200)
257      2200 FORMAT(1H1,5X," INTERNAL POINTS"/4X"COOR X",4X,"COOR Y",6X," FUNCTI
258      1ON")
259      WRITE(IMP,2201)(XINT(I),YINT(I),FINT(I),I=1,NP1)
260      2201 FORMAT(2F10.3,F15.3)
261      STOP
262      END
263 $

```